

Digital Braille Watch

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Manufacturing

· Custom gear and disk piece re-designed for low cost mass-production

Axle component serves to align disk and gears (Figure 8)

Abstract

The Braille system is the primary reading and writing method for 285 million visually impaired individuals worldwide^[1], yet no device exists that utilizes Braille to display time. Currently, to check time the blind use talking or tactile watches, but talking watches are disruptive, while tactile watches are difficult to read and fragile. The final Braille watch prototype utilizes a gear, disk and pin mechanism to display time. Preliminary testing verified that this design was accurate and easy-to-read, and force analysis identified a motor for use within the watch. By making molds of the watch components, it was proven that this design could be massproduced quickly and durably at low cost. The team has already submitted a patent application for the design and now aims to gain company interest and market the watch.

Background

Braille Basics^[2]

- · Method of written communication used by the visually impaired Numerical characters use a two-by-two grid Using different combinations of raised or lowered dots, all ten numbers
- can be displayed Size standards – dots at least 2.34 mm. apart, 0.48 mm. in height;
- characters 6 22 mm anart

Braille Numerals (Figure 1)

• • • • • • 0 0 0 0 2 3 4 5 6 7 8 9 1

Figure 2: Fall 2010

spring mechanism that keeps pins

design shows

below watch

Figure 1: The Braille numbers 0-9 each consist of four dots

Design Criteria

Improve upon competition

- Talking watch disruptive Tactile watch – fragile and difficult to read
- Several concept watches

Create an innovative and functional Braille watch

- Size of a standard wristwatch
- Silent, accurate, reliable
- Standard Braille numbering and spacing
- Advance past designs (Figure 2,3)

<u>References</u>

- [1] "Visual Impairment and Blindness". World Health Organization. Apr. 2011. 26 Apr. 2011. http:// www.who.int/media centre/factsheets/fs282/en/. [2] "Size and Spacing of Braille Characters." Braille Authority of North America. n.d. 27 Jan.
- 2010. <http://www.Brailleauthority.org/sizespacingofBraille

Final Design

Force Analysis

Figure 6: As the disk rotates, the pin applies a resisting force, F_{app}, to the disk

τ_{D1.D}

τ_{D2,D1}

Figure 7: By modeling the torque each gear exerts on

its neighboring gear, the maximum force required at the

drive gear can be determined

Disk 1

 ΣF_x : $F_{app} = N\cos 45^\circ + F_p \cos 45^\circ$ ΣF_v : $(W + kx) + F_p \cos 45^\circ = N\cos 45^\circ$

 $v_{pp} = (W + kx)(1 + \mu_p)$

 $1 - \mu_p$

Disk 2

T_{2,1} L T_{2,2}

 $\tau_{2,4}^{-}$

-d₂-

τ_{2.3} لأ لأ

 τ_{f2}

Design Concept

- Four rotating disks are located beneath watch surface. one for each Braille digit (Figure 4)
- Each disk has raised and recessed surfaces, which raise and lower pins (Figure 5) creating desired number
- Disks interact via integrated gears Overlapping the disks allows for
- standard spacing
- Multi-compartment casing holds watch together

Features

Calculations

- 12-hour watch with AM/PM indicator pin (up = PM)
- Rotating the drive gear once per minute will control the entire watch (Figure 4)
- Standard Braille spacing Final dimensions = 35.636 x 23.393 x
- 17.805 mm. (length x width x height).



Displays Displays Displays Displays 1-9; 0-2 0 or 1 0-5







raised and lowered in order to display the correct time

> Figure & Re-designing the custom gear and disk component into multiple parts allows for Molding Disks and Gears

- Molding provides low cost option for mass-production CNC milling used to produce disk master copy
- Viper si2 SLA printer used for remaining parts

Altering Design for Mass-Production

Gears were broken into layers

- Mold produced with silicone and catalyst (melting temperature = 340°C)
- Casting performed using urethane and various metal alloys

Testina

- Volunteers from the Wisconsin School for the Visually Impaired provided feedback on the prototype
- Spacing was ideal, easy to read
- Could be marketed to military and elderly in addition to the blind
- SolidWorks animation verified design accuracy

Future Work

Mass-Production

· Brass-casted parts, cast iron molds Implement motor (Figure 9)

Cost Analysis

- Material cost \$0.06 for brass mold
- Motor cost less than \$15

Marketing

- μ_m = friction between pin and disl d = distance of pin from center of disk (varies with disk and pin)

Torque on disk due to friction

x = compression distance of spring

Torque pin exerts on disk:

 $p_{isk\#,Pin\#} = \frac{\rho_{pin}V_{pin}g + kx}{1 - u}(1 + \mu_p)(d)$

 $1 - \mu_n$

 ho_{pin} = density of the pin (constant across all disks)

 V_{pin} = volume of the pin (varies with given disk)

k = spring constant of springs holding pins in place

 $r_{f,Disk\#} = \mu_c \frac{1}{3} \frac{\sigma_{ex}}{r_{out}^2 - r_{inv}^2}$ $\frac{inner}{2}(n(\rho_{pin}V_{pin}g + kx) + \rho_{gear}V_{gear}g)$

r____ = outer radius of base of gear shaft for given disk rinner= inner radius of base of gear shaft for given disk n = number of pins for the given disk

Torque applied by gears:

{D1onD2} =	$\left(\tau{2,1}+\tau_{2,2}+\tau_{2,3}+\tau_{2,4}+\tau_{f2}+\tau_{D3onD2}\right)$	
$D_{2onD1} =$	$\frac{\left(\tau_{2,1} + \tau_{2,2} + \tau_{2,3} + \tau_{2,4} + \tau_{f2} + \tau_{D3onD2}\right)}{d} (d_1)$)

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•Wisconsin School for the Visually Handicapped Proto Labs Inc. •Wisconsin Alumni Research Foundation •Callie Bell, J.D

 Diverse market BrailleNote etc.)

Manufacturing cost will continue to decrease with mass-production



- toraue of 3 6X10-3Nm
- Patent pending, sponsored by Wisconsin Alumni Research Foundation
- Technology has additional applications (elevator, low cost alternative to
 - Acknowledgements





Figure 9: A

micromotor will

he used to drive



Figure 3: Spring 2011 design demonstrates how the gear system within the Braille watch leads to intermittent rotation of the disks

Motor

- T_{max} well within range of many micromotors Safety factor > 2
- Faulhaber, Audemars, etc.

gears and casing All other forces were modeled as negligible

Modeled worse case

needed at drive gear

Considered 2 forces:

scenario to determine τ_{max}

· Force pin exerts on disk

slopes (Figure 6)

Friction force between

- Modeling torque relationship between all disks/gears gave Tmax =
- 3.6X10-3Nm (Figure 7)

- Various manufacturers –