



DIGITAL BEAM ATTENUATOR



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CLIENT: DR. CHARLES MISTRETTA

ADVISOR: DR. CHRIS BRACE

ABSTRACT

Our client, Dr. Charles Mistretta, is researching the use of dynamically attenuated X-Ray beams during computed tomography (CT) scans. Dynamic attenuation has been shown to increase signal to noise ratio uniformity and decrease X-Ray scatter^[3]. This improves image quality and reduces the necessary imaging dose^[3]. Our client has developed a prototype to test digital beam attenuation (DBA) and asked us to design a system to automate the prototype motion. Our final design uses a linear stepper motor to actuate one wedge. We found the wedge travels with a constant acceleration and moves with 10173 μ steps/mm. As the motor is currently too wide to actuate multiple wedges, future designs will require a smaller stepper motor.

INTRODUCTION

Client: Dr. Charles Mistretta, UW-Madison

- Departments: Medical Physics, Radiology, Biomedical Engineering
- Research: MRI and X-Ray Computed Tomography (CT)
- Project proposal: Mechanize a device used to test DBA

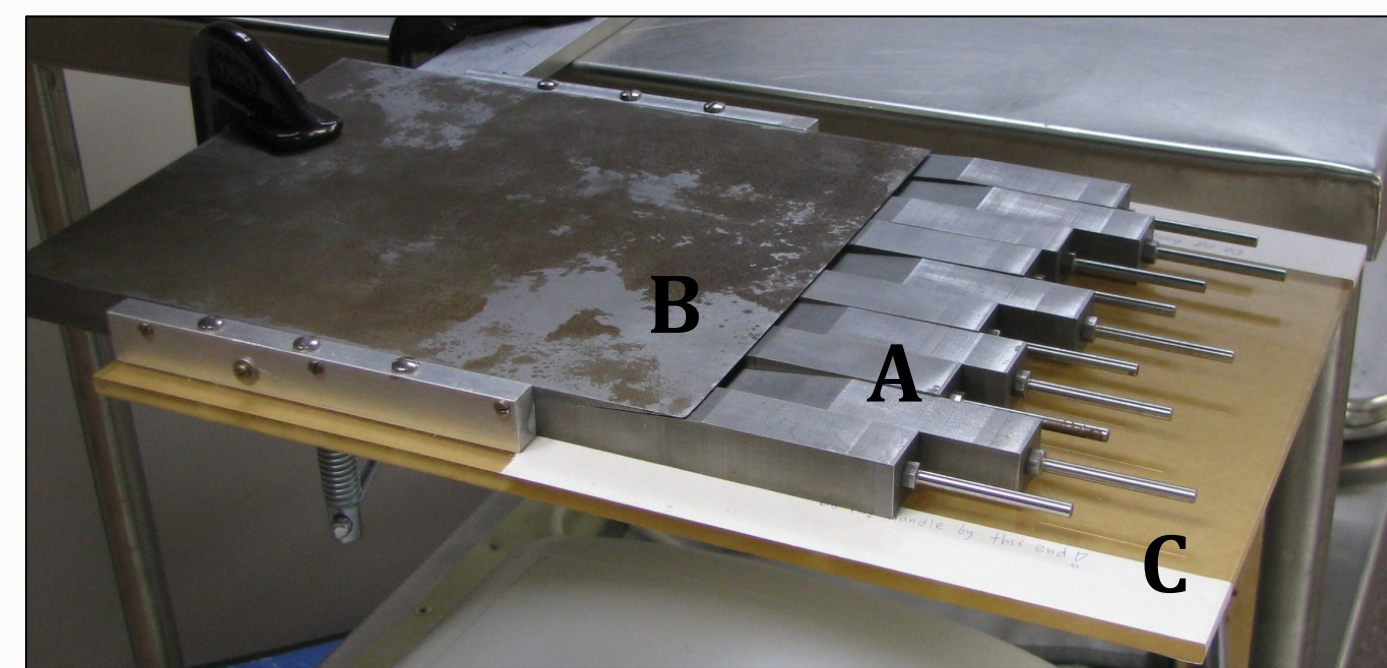


Figure 1: DBA Prototype
The prototype consists of ten hand-movable steel wedges (A) and a steel solid wedge (B). The prototype is mounted to a sheet of Plexiglas (C).
Photo Credit: Tim Szczykutowicz

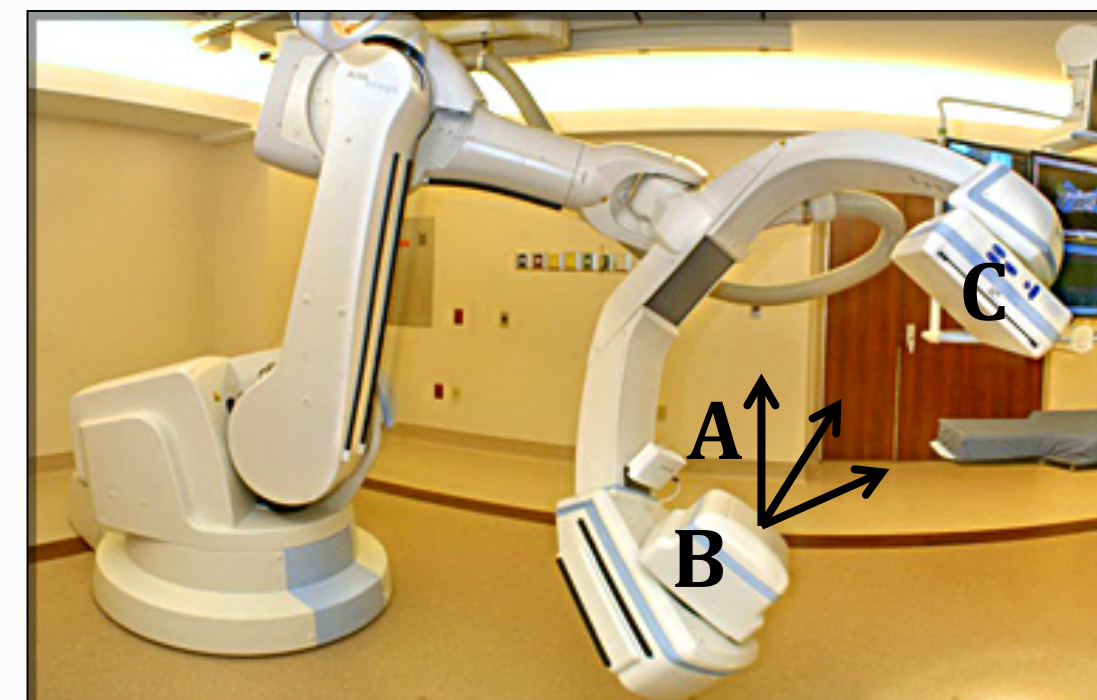


Figure 2: C-Arm CT Scanner
This scanner emits a fan-shaped array of X-Ray beams (A) from the source (B) and collects the X-Rays at the detector (C). It has a range of motion of 220 degrees. Photo Credit: Katherine Lake

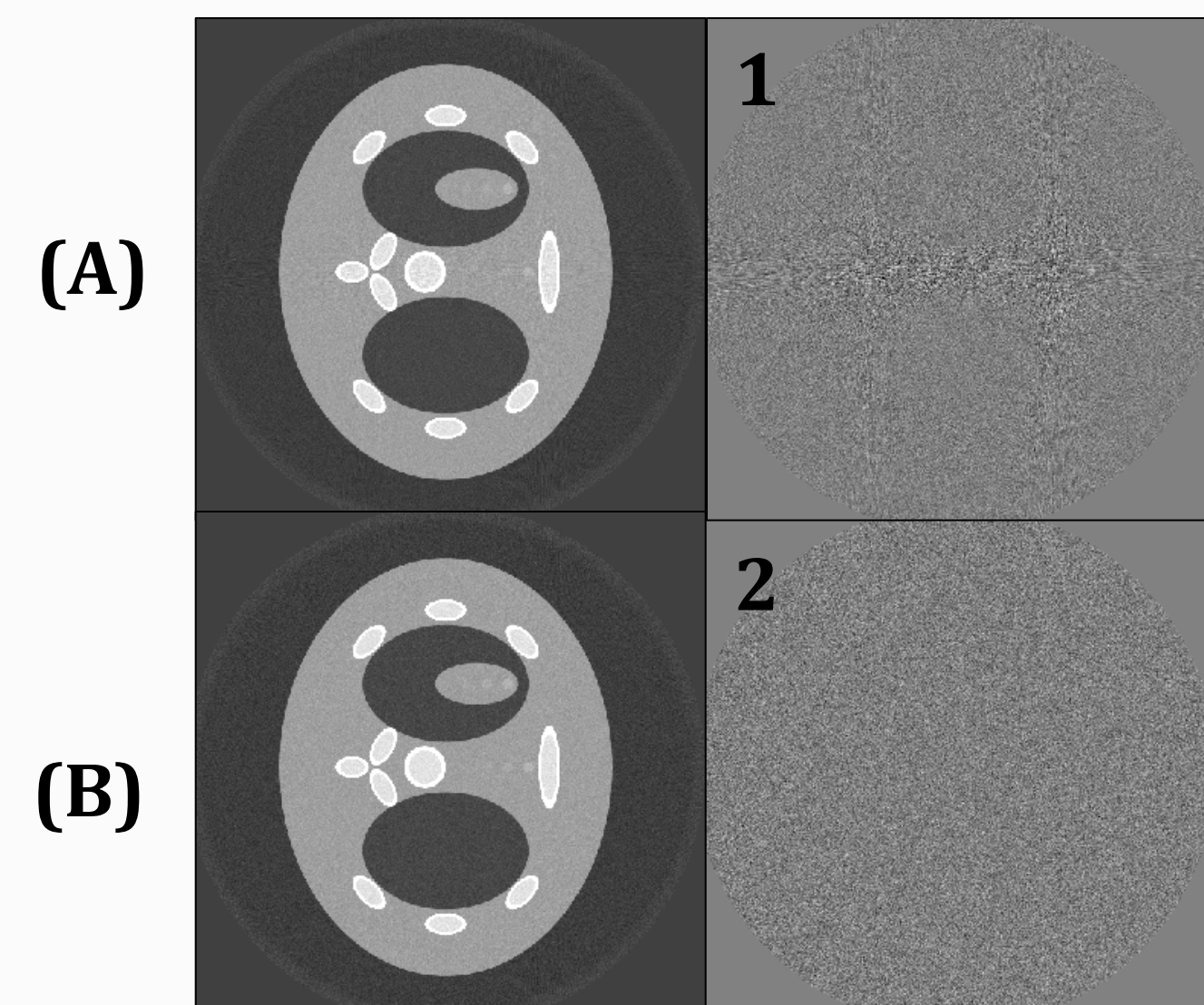


Figure 3: CT Phantom Scans (A),(1) Without DBA and (B),(2) With DBA
The DBA simulation uses 1240 leaves and the dose was 22% lower than the non-DBA simulation dose. (1) and (2) show reconstruction noise; (2) shows the preferred uniform noise of DBA images.
Photo Credit: Tim Szczykutowicz

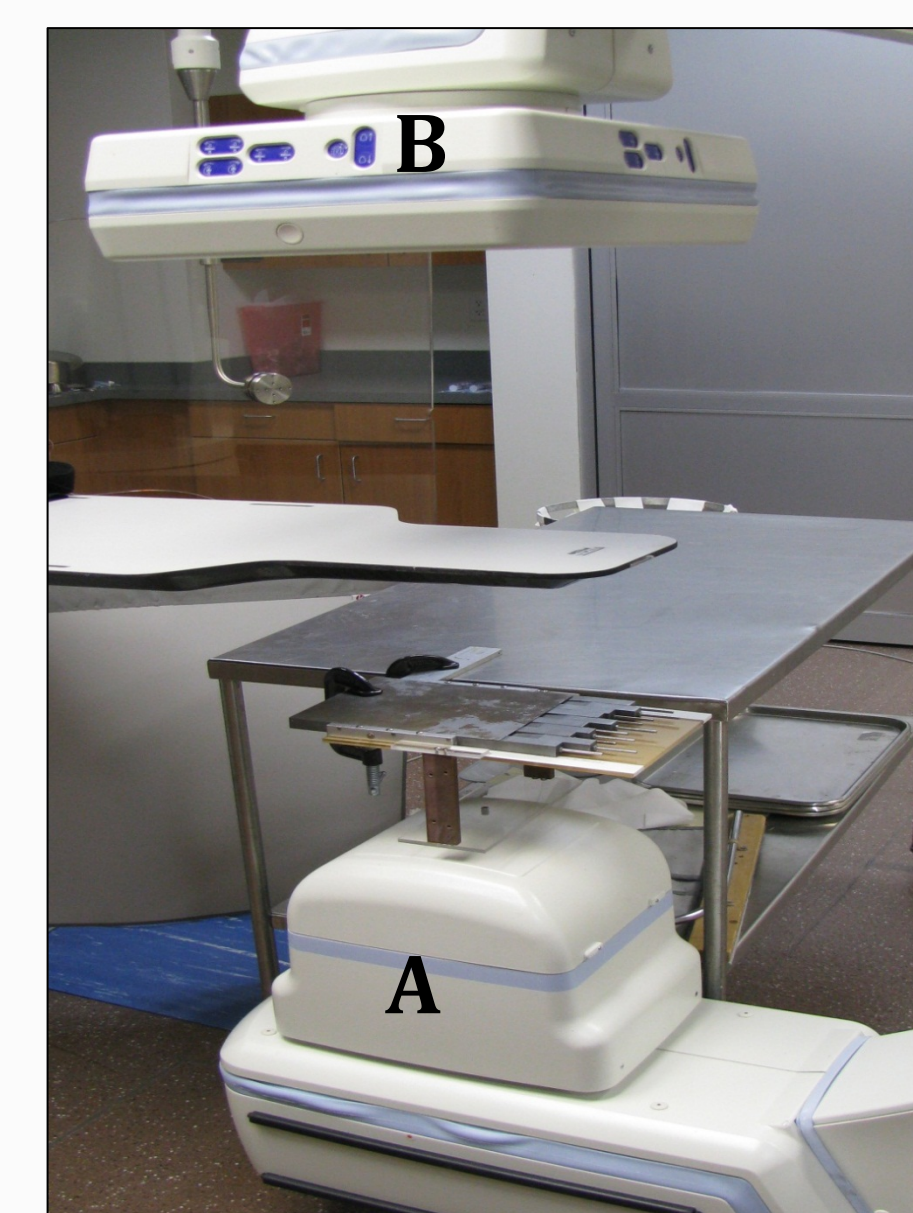


Figure 4: Mounted DBA Prototype
The prototype is mounted between the X-Ray source (A) and detector (B) to attenuate the X-Ray beams.
Photo Credit: Tim Szczykutowicz

DESIGN CRITERIA

- Scalable actuation method
- Each wedge:
 - Move independently of other wedges
 - Attain pre-programmed positions dependent on time
 - Maximum 1mm movement increments
 - Stroke length > 4cm
 - Minimum speed: 15 mm/s
 - Flush with base plate and neighboring wedges
- Post-scan report of individual wedge position and corresponding time

TESTING

SCALABILITY ANALYSIS



Figure 5: Micro Linear Motor (A) and Micro Stepper Motor (B)

Both are available commercially from Micromo. The linear motor produces linear motion and the stepper motor can be used with a lead screw to generate linear motion.
Photo Credit: [4], [5]

Findings	
Wedge Width	17.5 mm
Current Motor Width	56.4 mm
Linear Motor Width ^[2]	12.5 mm
Stepper Motor Diameter ^{[1],[2]}	6-12 mm

VELOCITY

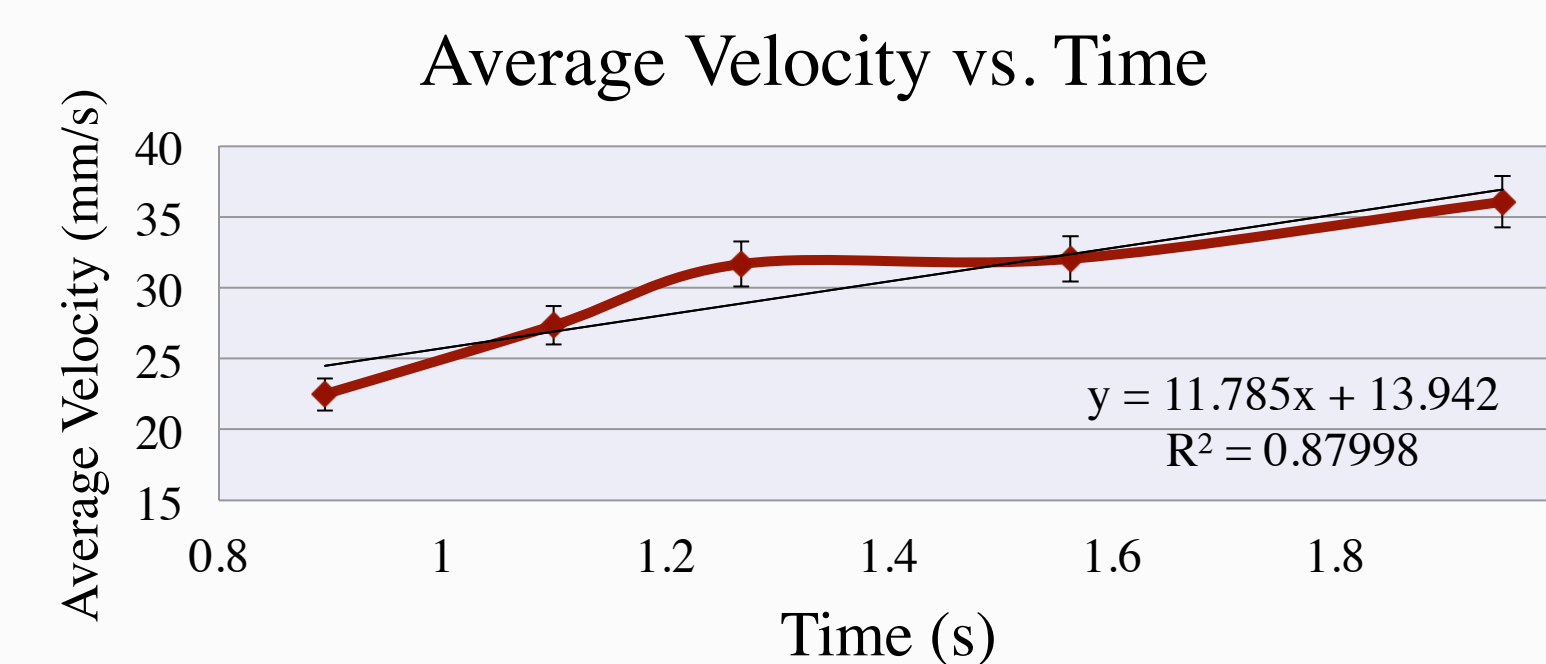


Figure 6: Velocity Testing Results

The time required to travel a known distance was measured and n = 10 trials were used to calculate the average velocity at each distance. The motor achieved an average velocity of 36 mm/s at d = 70mm and the estimated acceleration, using the slope of a linear trendline, is 11.78 mm/s².

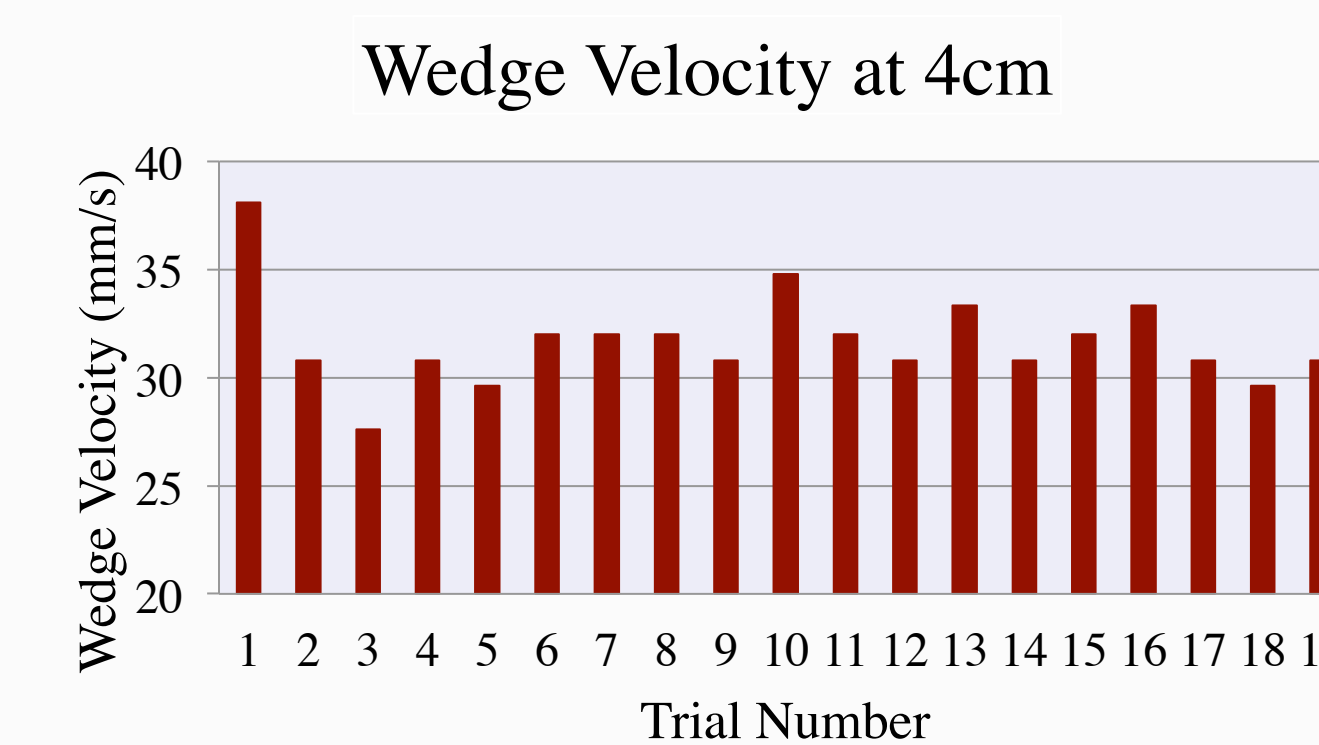


Figure 7: Wedge Velocity at Stroke Length
The average velocity at the stroke length of 4cm is 31.67 mm/s with a standard deviation of 2.15 mm/s.

FINAL DESIGN

Accomplished:

- Used Mdrive 23 plus linear actuator^{[1],[2]}
- Utilized scalable actuation method
- Independently actuated one wedge to desired specifications

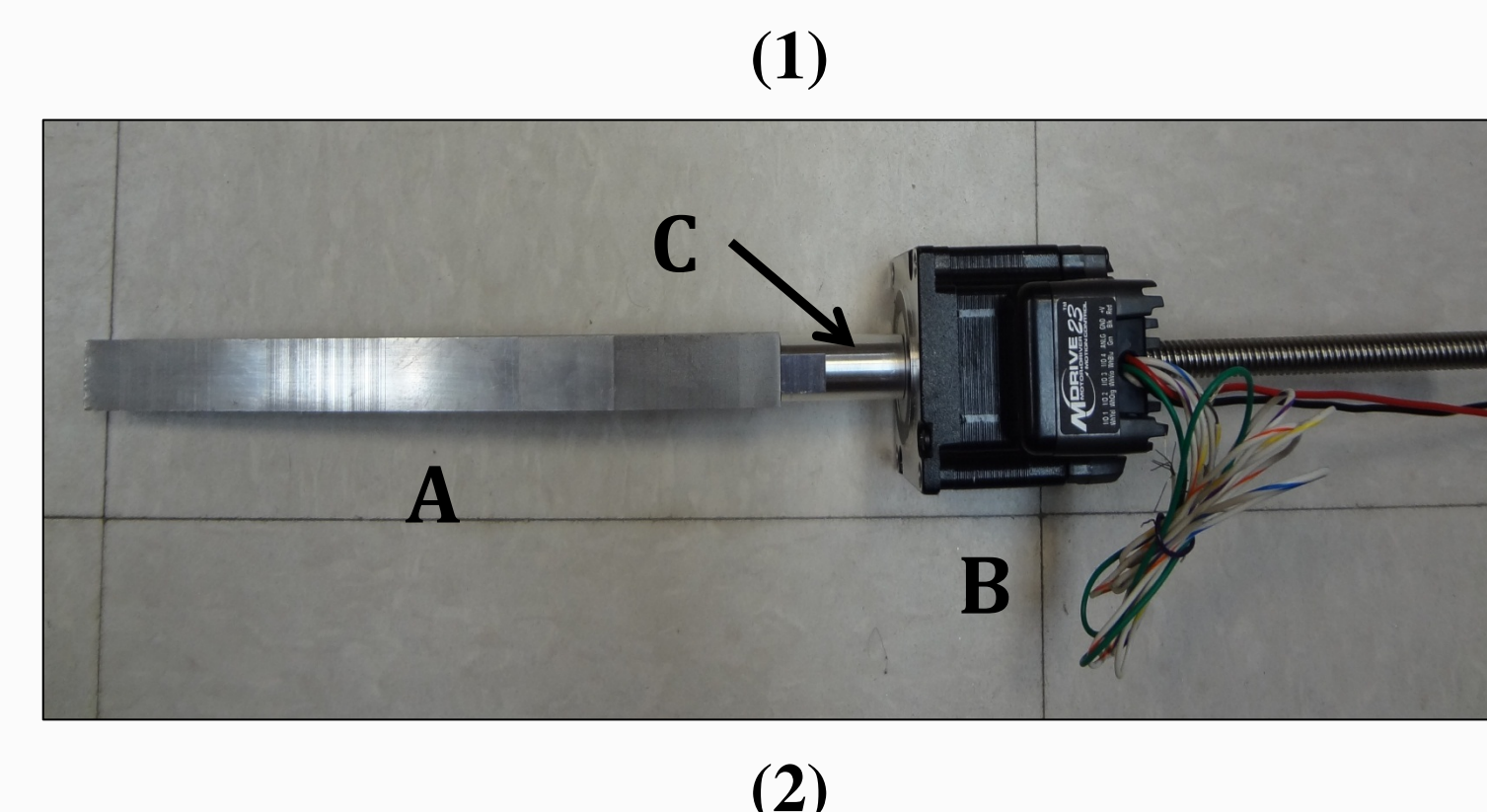
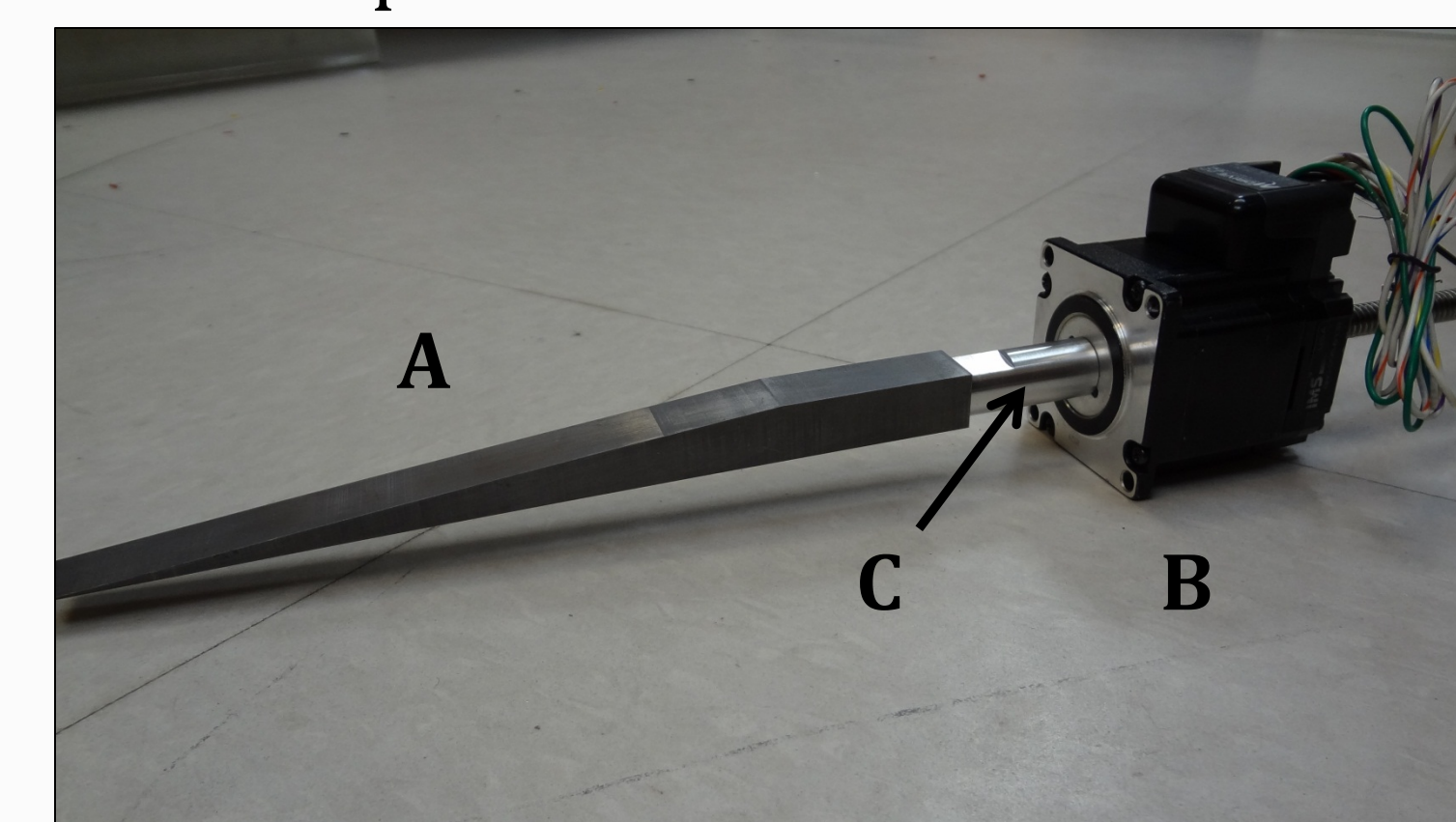


Figure 8: Single Wedge with Motor Attachment (1): Side View (2): Top View

The M-Drive 23 Plus linear actuator (B) is connected to the wedge (A) with an adapter (C) and can drive the wedge a 4cm stroke at a velocity of 31.67 mm/s.
Photo Credit: Katherine Lake

Movement Characteristics

- Initial Velocity 25 mm/s
 - Satisfies 15mm/s requirement
- Maximum stroke length 10 cm
 - Satisfies 4cm requirement
- Minimum movement increment 7.5E-5 mm
 - Satisfies maximum 1mm requirement

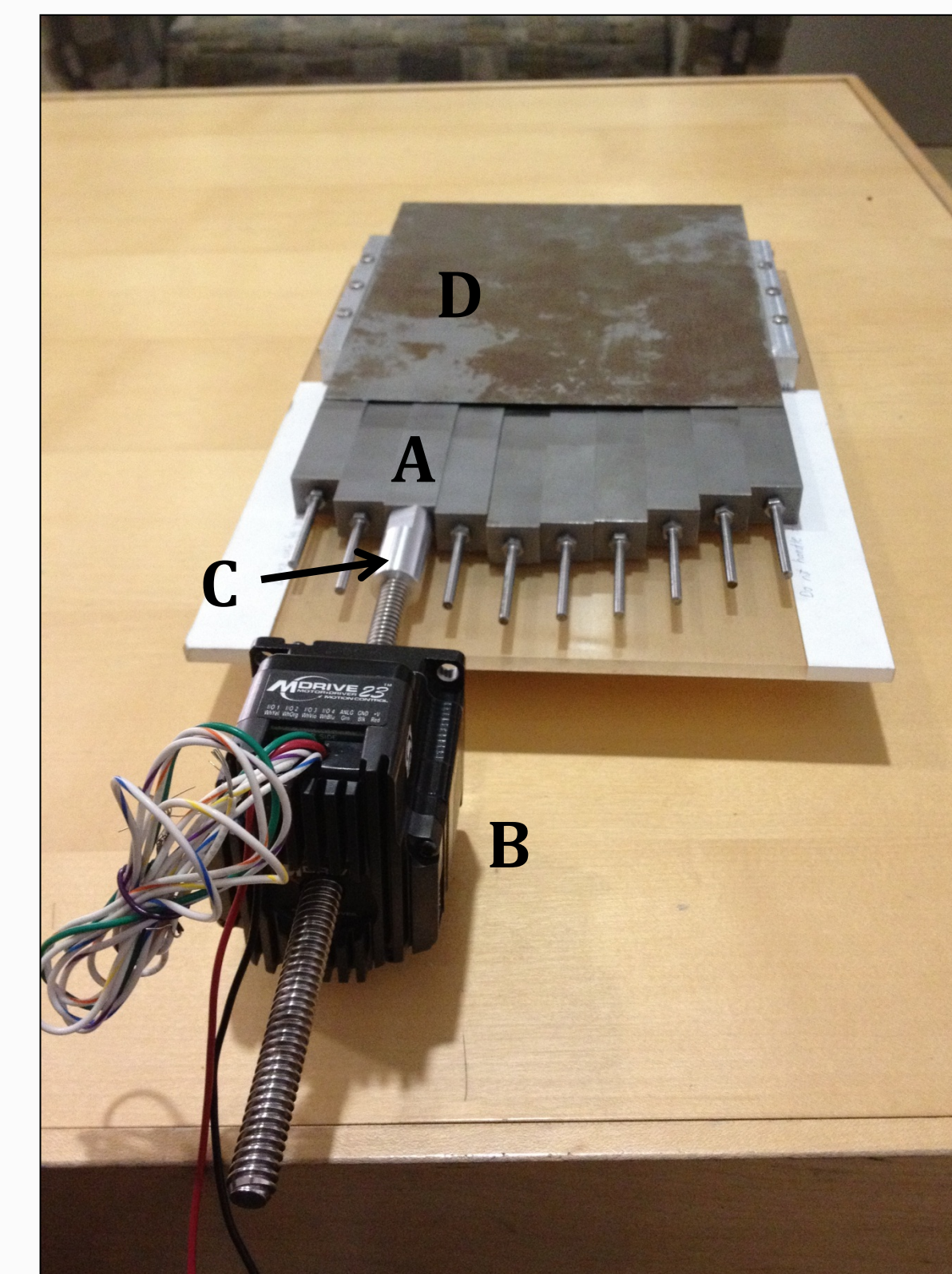


Figure 9: Actuated Prototype
The motor (B) drives the actuated wedge (A) inside the prototype housing (D) with a stroke length of up to 10 cm. The wedge is connected to the motor via an adaptor (C).
Photo Credit: Katherine Lake

POSITION

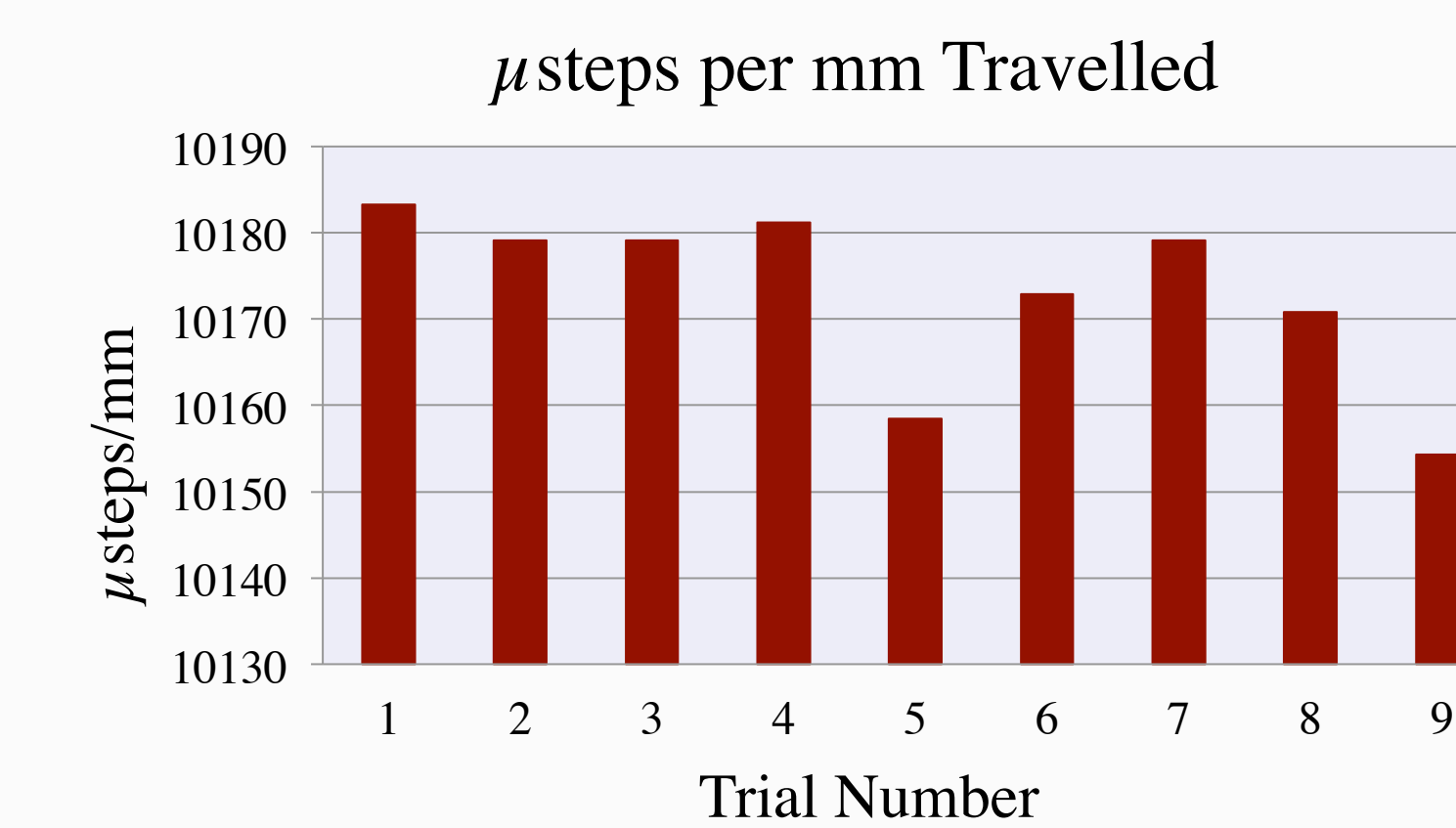


Figure 10: Motor Positional Precision
The motor can be precisely positioned using the integrated controller and lead screw. Nine trials using 500,000 μ steps/trial were completed.

Experimental Data	
Average measured μ steps per mm	10173 ^[2]
Actual μ steps per mm	10078
Standard Deviation	9.72 μ steps
Average Percent Error	.9356 %
Average Movement Increment	.026 mm/step

FORCE

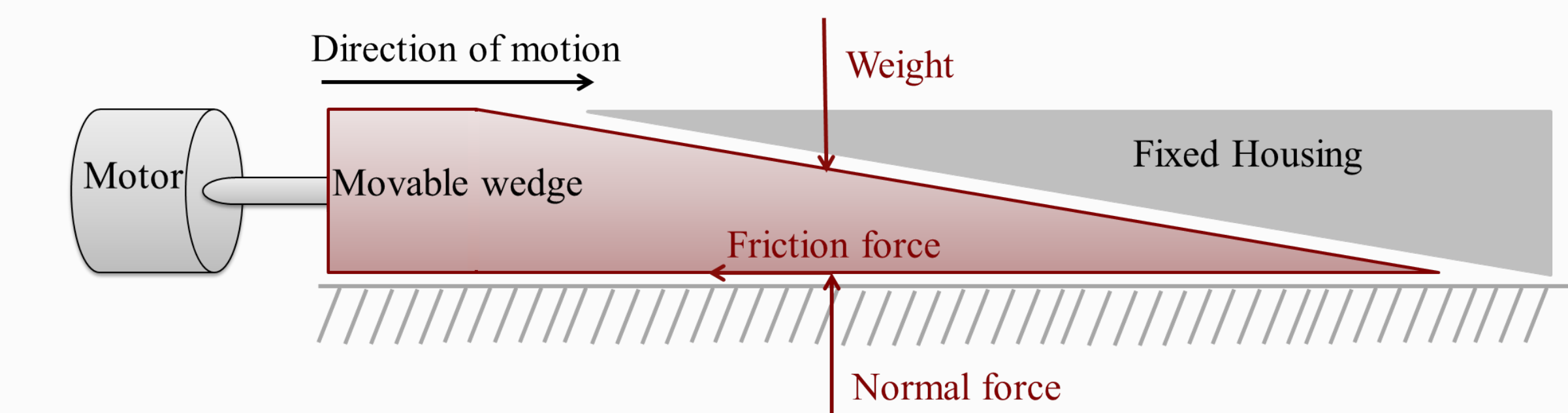


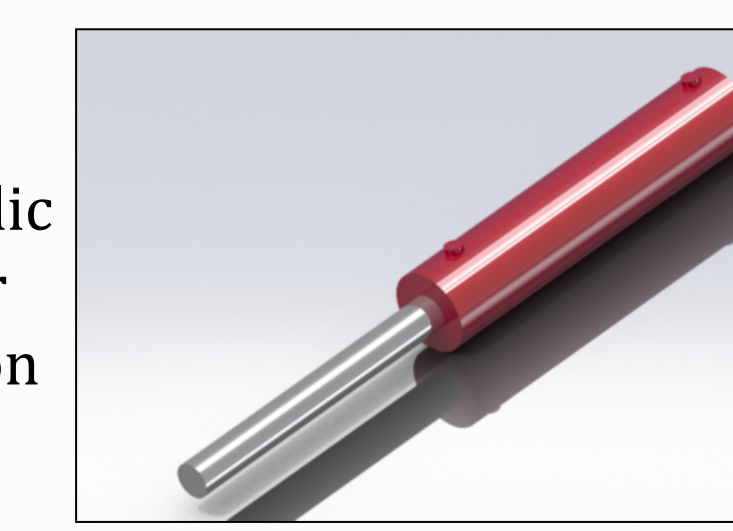
Figure 11: Wedge Free Body Diagram

A free body diagram was used to estimate the friction forces experienced by the wedge during motion. It assumes an additional normal force caused by the motor torque and negligible interactions with neighboring wedges. Using $\mu_s = 0.4$ between the 275g steel wedge and the Plexiglas base, the calculated static friction force is $F = 1.08N$.

FUTURE WORK

Figure 12: Hydraulic Cylinder

In some cases, micro hydraulic cylinders are more suited for applications of precise motion control in small spaces.
Photo Credit: [6]



- Actuate all wedges
- Increase number of wedges
- Utilize smaller motors
- Sinusoidal wedge motion
- Investigate hydraulics

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